

a)

b)

Department: Production and Mechanical Design Total Marks: 100 Marks



Course Title: Theory of Mechanical Vibrations

Date: Jan 2014 (First term Exam)

Course Code: MPD3115

Allowed time: 3 hrs

Year: 3<sup>rd</sup>
No. of Pages: (2)

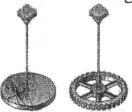
Remarks: (answer the following questions... assume any missing data... answers should be supported by sketches)

Problem number (1) (30 Marks)

Use the energy method to find the natural frequency of oscillation of the homogeneous cylinder as shown below. (8 Marks)

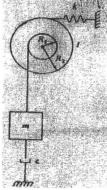


The disk and gear unlergo torsional vibration with the periods shown. (10 Marks) Assume that the moment exerted by the wire is proportional to the twist angle. Determine a) the wire torsional spring constant, b) the centroidal moment of inertia of the gear, and c) the maximum angular velocity of the gear if rotated through 900 and released. ( $\tau n = 1.13 \text{ s of desk}$ ), ( $\tau n = 1.95 \text{ of gear}$ ) and  $\tau = 6 \text{ disk} = \text{gear} = 20 \text{Ib}$ .



The amplitude of vibration of the system of figure shown decays to half of the initial value in 11 cycles with a period of 0.3 sec. Determine the spring stiffness and the viscous damping coefficient. Where I =2.4 kg.m², m = 5 kg, R1= 20 cm, R2 =40 cm.

(12 Marks)



Problem number (2)

(15 Marks)

A machine of 100 kg mass has a 20 kg rotor with 0.5 mm eccentricity. The mounting springs have k=85 kN/cm and the damping is negligible. The operating speed is 600 r.p.m and the unit is constrained to move vertically. Determine:

i) The dynamic amplitude of the machine.

ii) Redesign the mounting so that the dynamic amplitude is reduced to one half of the original value if the same natural frequency is maintained (i.e., find new M and k).

### Problem number (3)

(15 Marks)

A rotor of speed of a turbo-supercharger of 6 kg mass is keyed to the centre of 2 cm diameter steel shaft ( $E = 2 \times 10^6 \,\text{Kgf/cm}^2$ ). The distance between bearings is 30 cm. Assuming the shaft to be simply supported, determine:

- i) The critical speed.
- ii) The amplitude of vibration of rotor at a speed of 600 r. p. m. if eccentricity is 0.002 cm.
- iii) The maximum vibration force transmitted to the bearings.

### Problem number (4)

a)

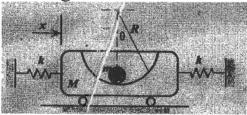
b)

(25 Marks)

- A vehicle has a mass of 1800 kg, the wheel base is 3.6 m, the cg is 1.6 from front axle, and the radius of gyration about cg is 1.4 m. If the spring constants of front and rear are 42000 and 48000 N/m respectively, determine:
  - i) The two natural frequencies of the system.
  - ii) The principle modes of vibration.
  - iii) The motions x(t) and  $\theta(t)$ .

(13 Marks)

Using Lagrange's equations to drive the equation of motions of the system shown in the figure below. The circular cylinder has a mass m, radius r, and rolls without slipping inside the circular groove of radius R. (12 Marks)

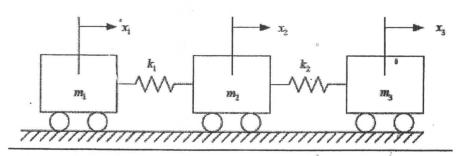


Problem number (5)

(15 Marks)

In the system shown below in Figure, the displacements x1, x2 and x3 are measured from the static equilibrium position of the system. Write a Matlab program to compute the eigenvalues and eigenvectors (modes shapes) of the system.

Assume that k1 = k2 = k = 1N/m, and m1 = m2 = m3 = m = 1 kg.







### Department: Production and Mechanical Design Total Marks: 100 Marks



Faculty of Engineering

Course Title: Theory of Metal Cutting

Course Code: MPD3115

Year: 3<sup>rd</sup>

Date: Jan 2014 (First term Exam)

Allowed time: 3 hrs

No. of Pages: (1)

Remarks: (answer the following questions... assume any missing dat 1... answers should be supported by sketches)

Prob	lem number (1)	(20 Marks)		
What is t Distingui Explain t	he geometry of single point he importance of rake ang sh between an oblique and he dimension of chip of ol the variable affect on the	le? I an orthogonal in metal blique and orthogonal in	cutting? metal cutting?	(4 Marks) (4 Marks) (4 Marks) (4 Marks) (4 Marks)
<u>Prob</u>	lem number (2)	(30 Marks)	5	
What are		ntions? 1.25mm/rev, and chip th		(5 Marks) (5 Marks) s 2mm; determine
The tool bi speed = 30	<ul> <li>i) Chip thickness ratio</li> <li>ii) Shear angle.</li> <li>thas a rake angle of 10°. I m/min., and coefficient of</li> <li>i) Shearing force.</li> <li>ii) Friction angle.</li> <li>iii) Cutting force.</li> <li>iv) Horse power at the or</li> </ul>	f shear strength 6000 kg f friction = 0.9. Determi	y/cm², width of cut = 1 ne the following:	
Prob	em number (3)	(25 Marks)		(20 Marks)
Distinguis Explain th What are A carbide 50 m/min b	sh between carter and flance cutting temperature and the factors of tool selection cutting tool, when machinated for 100 minutes. Detwhat speed the tool is to learn	k wear? its importance? n? ned with mild steel worl ermine the life of the to	ol when the cutting sp	peed is increased
Proble	m number (4)	(25 Marks)		
Explain the Explain the	the types of the cutting too the desired properties of cut the methods to separate the required to cut a certain	tting fluids? cutting fluids and chips	?	(5 Marks) (5 Marks) (5 Marks)

coolant with specific heat 0.80 and specific gravity 0.85 which conducts away 0.75 of the heat produced, determine the rise in the temperature of coolant due to this cut in F° and C°. (10 Marks)



a)

b)

a)

b)

c)

a)

c)

d)

Department: Production and Mechanical Design
Total Marks: 60 Marks



Course Title: Machining Machines

Course Code: MPD3118

Year: 3<sup>rd</sup>

Date: Jan. 2014 (First term)

Allowed time: 3 hrs

No. of Pages: (1)

Remarks: (answer the following questions... assume any missing data... answers should be supported by sketches)

	Problem number (1) (15 Marks)	0			
a)	Classify the machine tool parts?			(4 Marks)	
b)	What are the main types of ribbing systems?			(3 Marks)	
c)	What are the types of chip storage of machine tool?			(4 Marks)	
d)	What are the factors affect the accuracy of machine tool?		4	(4Marks)	
	Problem number (2) (15 Marks)	4	j.		
8			12		

Classify the different types of joints of the machine tools and its importance? (5 Marks) A joint cantilever structure with end point load of 100 N at 120 mm distance from the joint. If the solid part has 20 mm in diameter, assume  $I_s/I_j$ =4/9, E=21000 Kgf/mm², and joint surface m=  $2000mm^{-1}$ . A) What is the required interface pressure and bolt size to have joint deflection of 25  $\mu$ m, while the joint stiffness should not be less than  $15x10^6$  N/mm. Note that the available bolt sizes Areas of bolt mm², M6, 20.1, M8, 36.6 and M10, 58.6 respectively. B) Compute the joint diameter which fulfils the above requirements. (10 Marks) Useful equation:

For jointed cantilever with end point load;  $\delta_j = \frac{WL^2}{mP_m I_j}$ 

#### Problem number (3) (15 Marks)

Mention the requirements made to machine tool? (5 Marks)

What the ways to isolate vibrations of machine tool? (5 Marks)

Machine tool having a dynamic load of 7 N. The logarithmic damping decrement ( $\Delta$ ) 0.13. From power spectrum the amplitude A/ $\sqrt{2}$  =7.07 $\mu$ m at the band width ( $\Delta\omega$ ) is 40 Hz. Calculate the dynamic stiffness and natural frequency of this machine. (5 Marks)

## Problem number (4) (15 Marks)

Mention the rules should take when set up machine tools on the foundation? (3 Marks)

b) What are the geometrical and the practical test of machine tools? (3 Marks)

Distribute the forces act in the radial drill? (3 Marks)

A machine tool having a moving column with 300 Kg weights along the machine bed. The bed is 8.0 m long, 1.2 m width; flexural rigidity  $EI_1 = 24 \times 10^6$  Kgf/m². What is the suitable concrete depth to keep the deviation from the alignment of the bed/foundation system not more than  $6 \times 10^{-5}$  m measured between the mid end (a=0) and its end (a=1/2). Take the concrete width is equal to the bed width, and the concrete modulus of elasticity as  $24x10^8$  Kgf/m². (6 Marks)

Useful equations:

$$(EI)_3 = (EI)_1 + (EI)_2$$

The deviation from alignment is given as,

$$\Delta = \frac{W_m l^3}{(EI)_3} \left[ -\frac{1}{16} \left( \frac{a}{l} \right)^2 + \frac{1}{6} \left( \frac{a}{l} \right)^3 + \frac{5}{24} \left( \frac{a}{l} \right)^4 - \frac{1}{10} \left( \frac{a}{l} \right)^6 \right]$$





### TANTA UNIVERSITY **FACULTY OF ENGINEERING**



1/2

DEPARTMENT OF: Production Engineering & Mech. Design Dep. EXAMINATION (3 VEAR) STUDENTS OF Production ENGINEERING

	COURSE TITLE: Mech	anical Design (2)	,1	COURSE CODE: PMD32114	
DA	ATE:22- 01-2014	TERM: First term	TOTAL ASSESSMENT MARKS: 75	TIME ALLOWED: 3 HOURS	

Notes:

الامتحان مكون من 3 أسئلة في صفحتين وورقتين

It is allow for student to use bearing table and only one text book Systematic arrangement of calculations and clear neat drawings are essential. Any data not given is to be assumed – Answer as many questions as you can. Answer as brief. as possible.

### PROBLEM # ONE (40%)

- What types of bearings do you chose for a turbine running at 100 RPS Iwith a rotor weight 250 kN and shaft diameter 30 cm? Justify your answer.
- A sleeve of journal bearing is 20 mm long and 40 mm in diameter. It IIhas a clearance of 0.04 mm and uses SAE 20 lubricant at an operating temperature of 60°C. The bearing supports a load of 180 Kg<sub>f</sub>. Calculate the heat generated for speeds of 1000, 2000, 3000, 4000 RPM, and construct a graph of the results.

### PROBLEM # TWO (50 %)

- Why are cylindrical rolling bearings used? What magnitude of thrust <u>I-</u> load can they withstand?
- A spindle for a milling machine is to be supported by two bearings. II-The end mill holder (located outside of the bearings supports as shown in figure (1)) is to rotate at 4000 RPM. Under the most severe conditions, the bearing (A in the figure (1)) closest to the end mill holder will experience a radial load of 2100 N and axial load of 1600N. The bearing (B) farthest from the end mill holder will experience only a radial load of 1800 N. If the shaft is subjected to a light shock loading and cannot exceed 60 mm diameter. Select suitable bearings for this operation. What is the rating life in hours for the bearings selected? Compare this life with the suggested hours of operation stated in the text book you have. Assume the machine work continuous for 8 hr/day. Draw with scale the suggested bearings arrangement with the bearings being separated by 15 cm.

### PROBLEM # 3 (30%)

- <u>I-</u> What is the major assumption in Lewis equation? And what is your opinion on them.
- II- Compeer between belts and gears in view of transmitting power?
- A sleeve of journal bearing is experiences a Sommerfeld Number of 0.25 and minimum film thickens of 0.01 mm and a radial clearance of 0.02 mm so what is your suggestion for the length to diameter ratio.

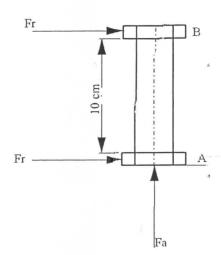


Figure (1)

Tanta University Faculty of Engineering Mechanical Power Eng. Dept. January 18, 2014 – 3<sup>rd</sup> Year



First Semester Exam Refrigeration & Air conditioning Time: 3 hours –Full Mark: [50]

Answer the following questions. Assume any necessary assumptions. Use of tables and charts of Refrigeration & Air conditioning is allowed.

Mark

1. a) What is the difference between a heat pump and a refrigerator?

[9]

- b) Discus and draw a domestic refrigerator.
- c) Define the following
  - 1- Dry Bulb Temperature
- 2- Wet Bulb Temperature
  - at one gainfind off 3- Relative Humidity amblind amissamma s at (notice the
    - 4- Specific Humidity
  - 2. The cock pit of a jet plane is cooled by a simply cooling cycle. Assuming [10] the following data, or say your bas also us obia sag (land but drion) all Plane speed =1200km/hr 2002 mm-001 bas notished mm-02 sond sonl

Ram efficiency = 90% and mm-d bas med 1 xm2 min drion gained wobally

Ambient air pressure= 0.85 ata.

Ambient air temperature= 30 °C. O.O. III m.A. OO.O. mod show a nosmod

Pressure ratio of the main compressor = 4

Temperature of air leaving the heat exchanger =60 °C.

Pressure in cock pit = 1 ata. See and other to the guideout sed to the belast.

Temperature in cock pit = 25 °C.

Isentropic efficiency of the main compressor is 75 % and of cooler turbine is 80%.

Load in the cock pit = 10 tons.

#### Determine:

- a) The quantity of air passed through the cooling turbine.
- b) COP of the cycle.
- A refrigerant 12 vapor compression system operating at 0.1 Mpa with [4] cooling capacity 25 T.R. Condensing pressure is 0.8 Mpa.

#### **Determine:**

- a) COP of the cycle.
- b) The mass flow rate of refrigerant in kg/s.
- c) Power of compressor in kW.

- 4. A vapor compression cycle utilizes 3 stores with capacities 50, 30, 40 T.R at -30, -20, and -10°C respectively. The system uses R-22 and has multiple expansion valves with flash chambers and multiple compression with flash intercooling. Condensing occurs at 45 °C. Assume no pressure drop. Draw the flow diagram and P-h chart of the cycle and determine
  - a) mass of refrigerant at each compressor.
  - b) heat rejected from the condenser.
  - c) COP of the cycle.
- 5. A room 8m×5m×3m height (with the longer wall oriented facing north direction) in a gymnasium building is to be conditioned. The building site is 32° North latitude. West wall separate the room from conditioned space otherwise south wall separates the room from unconditioned space and they are constructed from 100-mm face brick, 100-mm common brick. Other walls (north and East) are side streets and they are constructed from 100-mm face brick, 50-mm insulation and 100-mm concrete. The room has single window facing north with 2m×1.5m and 6-mm single glass having light color medium weave shading. The average number of occupants in space is 15 person's works from 9.00 Am till 9.00 Pm. Lighting is unvented and unsuspended fluorescent lamps, number of lamps are 8 and each lamp has 40 watt. Inside design condition is 25°C and outdoor air is assumed to be 40°C. Neglect all other loading and calculate the space total load and the sensible heat factor (SHF). Base your calculations on July, 3.00 Pm O'clock.

Good luck.

Dr. M. K. El-Fakharany

Faculty of Engineering

A refrigerant 12 vapor compression system operating at 0.1 Mpa with

cooling capacity 25 T.R. Condensing pressure is 0.3 Mpa.

Tanta University
Faculty of Engineering
Production Engineering & Design Dept.
3<sup>rd</sup> Year Prod. 2013/2014



Theory of Metal Forming Saturday, 4.01.2014 Time allowed: 3 hours Final Grade: 100 Marks

#### Final Exam

In deep drawing processes max initial

# Solve all questions and make use of the information given at the end:

# Question 1: (20 marks)

- a) Find the relation between engineering- and true-strain and the relation between engineering- and true-stress of rounded tensile specimen.
- b) A circle 1.5 cm diameter was printed on a sheet of metal with thickness 1 mm prior to a complex stamping operation. After the stamping, it was found that the circle had become an ellipse with major and minor diameters of 1.7 and 1.5 cm. Assume that the stress strain law is:

$$\overline{\sigma} = 500 \, \overline{\varphi}^{0.22} (MPa)$$

If the stress normal to the sheet is neglected and the material yields according to von-Mises criterion then determine:

- i- The principal strains.
- ii- The final thickness.
- iii- The effective strain.
- iv- The applied stress ratio  $\alpha = (\sigma_2/\sigma_1)$ .
- v- The final stresses in the element.
- vi- The total work done/unit volume in this forming process.

# Question 2: (25 marks)

ed to accomplish a wire

d with a yield strength of

a) A tension test is performed on a sample of steel 8 mm in diameter and gauge length of 8 cm and the following table shows the load and elongation counterparts:

Load (kg)	1390	1665	1920	2150
Elongation (mm)	1.21	2.53	3.52	14.25

Use the above data to get the constants "K" and "n" in the flow stress-strain relation ( $\sigma = K \varphi^n$ ) of the steel sample.

- b) Annealed rods from the steel tested in (a) of 25 mm in diameter are formed by extrusion. The billet has a diameter of 35 mm. The extrusion efficiency is 65%. Calculate:
  - i. The radial, tangential and longitudinal strains.
  - ii. The effective strain
  - iii. The extrusion force ( Derive the used law )
  - iv. The power required to extrude the rods at a rate of 0.5 m/s.

150 MPa and modulus of clasticity 200 GPa. On which radius should the bend

# Question 3: (20 marks)

- a) What are the main assumptions made for the calculation of the ideal drawing force in deep drawing processes
  - b) Cylindrical cups with a mean diameter of 150 mm and 250 mm height are deep drawn from sheets with thickness of 2 mm. One redrawing is necessary to produce the cup successfully. The first drawing ratio is 1.9
    - i- What is the blank diameter required to produce these cups?

ii- What is the redrawing ratio?

- iii- Calculate the ideal drawing force during the first drawing, when the depth of drawing is 0.3 its full value at this first drawing.( take  $\sigma_{fm} = 400$  MPa)
- iv- Assume the same mean flow stress, get the ratio of the maximum ideal drawing forces between the two stages.

# Question 4: (20 marks)

- a) Use the slab analysis to get the average pressure value acting on a slab of width "b" and height "h" and subjected to upsetting process.
- b) A tensile specimen has homogeneous properties but two regions of different areas. Get the limit strain for this specimen, if the material is only strain rate sensitive.
- c) The thickness of a sheet varies from 8.00 mm to 8.01 mm depending on location so tensile specimens cut from a sheet have different thicknesses. For a material with m = 0.05 and n = 0, what will be the strain in the thicker region when the strain in the thinner region is 0.5 and  $\infty$ .

a) A tension test is performed on a sample of si

of 8 cm and the following table shows the load

# Question 5: (25 marks)

- a) Derive an expression for the total drawing force required to accomplish a wire drawing process and hence find the optimum die angle.
- b) A sheet of thickness 20 mm and width 1 m is requested to be bent by 3-roll forming to form a tube of diameter 1 m. The sheet is made of steel with a yield strength of 350 MPa and modulus of elasticity 200 GPa. On which radius should the bend machine be configured?

# **Useful information:**

Flow rule: 
$$\frac{d\varphi_1}{(\sigma_1 - \sigma_m)} = \frac{d\varphi_2}{(\sigma_2 - \sigma_m)} = \frac{d\varphi_3}{(\sigma_3 - \sigma_m)} = \frac{3}{2} \frac{d\overline{\varphi}}{\overline{\sigma}} \qquad ; \qquad d\overline{\varphi} = \sqrt{\frac{2}{3} \left[ d\varphi_1^2 + d\varphi_2^2 + d\varphi_3^2 \right]}$$

von-Mises yield criterion:  $\sigma_{\gamma} = \frac{1}{\sqrt{2}} \sqrt{(\sigma_1 - \sigma_2)^2 + (\sigma_2 - \sigma_3)^2 + (\sigma_3 - \sigma_1)^2}$ 

For deep drawing:  $F_{ideal} = \pi d_m S \sigma_{f_m} \ln \left( \frac{D}{d_m} \right)$ ; For springback calculation:  $\left[ \frac{1}{\rho_0} - \frac{1}{\rho_1} \right] = 3 \frac{Y}{hE} - 4 \rho_0^2 \left( \frac{Y}{hE} \right)^3$